

Lista de Exercícios - 03

Observações:

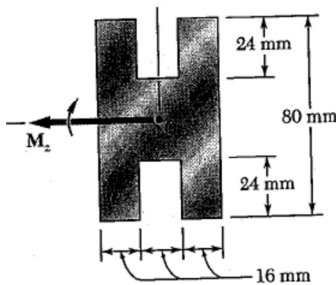
(a) A literatura e os exercícios abaixo indicados são considerados básicos. Recomenda-se que outros títulos também sejam consultados.

Literatura de Referência:

BEER, F. P., JOHNSTON, E. R., DEWOLF, J. T., MAZUREK, D. F. **Mecânica dos Materiais**. 5.ed. Porto Alegre: AMGH, 2011. 800p.

Flexão pura / reta: [p.244]

4.3 Uma viga com a seção transversal mostrada na figura é extrudada de uma liga de alumínio para a qual $\sigma_E = 250$ MPa e $\sigma_L = 450$ MPa. Usando um coeficiente de segurança de 3,00, determine o maior momento fletor que pode ser aplicado à viga quando ela é flexionada em torno do eixo z.



$$\text{Allowable stress} = \frac{\sigma_u}{F.S.} = \frac{450}{3} = 150 \text{ MPa} = 150 \times 10^6 \text{ Pa}$$

Moment of inertia about z axis.

$$I_1 = \frac{1}{12} (16)(80)^3 = 682.67 \times 10^3 \text{ mm}^4$$

$$I_2 = \frac{1}{12} (16)(32)^3 = 43.69 \times 10^3 \text{ mm}^4$$

$$I_3 = I_1 = 682.67 \times 10^3 \text{ mm}^4$$

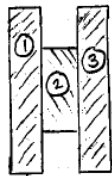
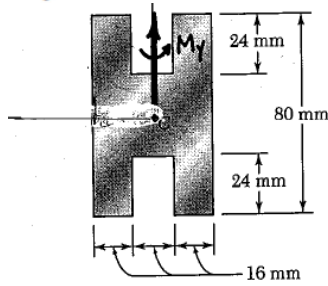
$$I = I_1 + I_2 + I_3 = 1.40902 \times 10^6 \text{ mm}^4 = 1.40902 \times 10^{-6} \text{ m}^4$$

$$\sigma = \frac{Mc}{I} \quad \text{with } c = \frac{1}{2}(80) = 40 \text{ mm} = 0.040 \text{ m}$$

$$M = \frac{I\sigma}{c} = \frac{(1.40902 \times 10^6)(150 \times 10^6)}{0.040} = 5.28 \times 10^3 \text{ N}\cdot\text{m} \quad M = 5.28 \text{ kN}\cdot\text{m}$$

4.4

4.4 Resolva o Problema 4.3 considerando que a viga seja flexionada em torno do eixo y.



$$\text{Allowable stress} = \frac{\sigma_u}{F.S.} = \frac{450}{3.00} = 150 \text{ MPa} = 150 \times 10^6 \text{ Pa}$$

Moment of inertia about y-axis.

$$I_1 = \frac{1}{12}(80)(16)^3 + (80)(16)(16)^2 = 354.987 \times 10^3 \text{ mm}^4$$

$$I_2 = \frac{1}{12}(32)(16)^3 = 10.923 \times 10^3 \text{ mm}^4$$

$$I_3 = I_1 = 354.987 \times 10^3 \text{ mm}^4$$

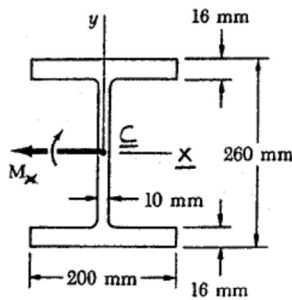
$$I = I_1 + I_2 + I_3 = 720.9 \times 10^3 \text{ mm}^4 = 720.9 \times 10^{-9} \text{ m}^4$$

$$\sigma = \frac{M_e}{I} \quad \text{with} \quad c = \frac{1}{2}(48) = 24 \text{ mm} = 0.024 \text{ m}$$

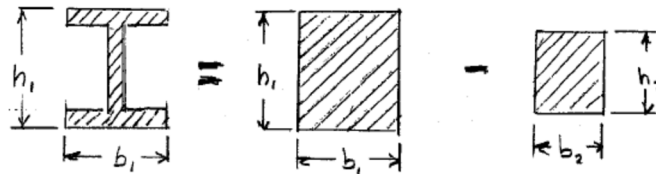
$$M = \frac{I\sigma}{c} = \frac{(720.9 \times 10^{-9})(150 \times 10^6)}{0.024} = 4.51 \times 10^3 \text{ N}\cdot\text{m} \quad M = 4.51 \text{ kN}\cdot\text{m}$$

4.5

4.5 A viga de aço mostrada é feita com um aço de grau tal que $\sigma_E = 250 \text{ MPa}$ e $\sigma_L = 400 \text{ MPa}$. Usando um coeficiente de segurança de 2,50, determine o maior momento fletor que pode ser aplicado à viga quando ela é flexionada em torno do eixo x.



The moment of inertia I_x is equivalent to that of a rectangle with a cutout



Larger rectangle: $b_1 = 200 \text{ mm}$ $h_1 = 260 \text{ mm}$ $I_1 = \frac{1}{12} b_1 h_1^3$

$$I_1 = \frac{1}{12}(200)(260)^3 = 292.933 \times 10^6 \text{ mm}^4$$

Smaller rectangle: $b_2 = 200 - 10 = 190 \text{ mm}$ $h_2 = 260 - (2)(16) = 228 \text{ mm}$

$$I_2 = \frac{1}{12}(190)(228)^3 = 187.662 \times 10^6 \text{ mm}^4$$

Section: $I_x = I_1 - I_2 = 105.271 \times 10^6 \text{ mm}^4 = 105.271 \times 10^{-6} \text{ m}^4$

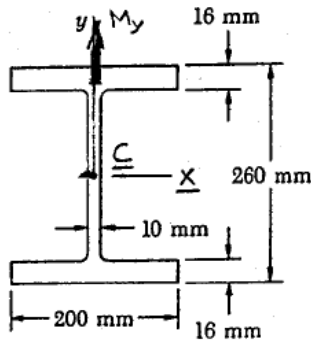
$$c = \frac{260}{2} = 130 \text{ mm} = 0.130 \text{ m}$$

$$\sigma_{all} = \frac{\sigma_u}{F.S.} = \frac{400}{2.50} = 160 \text{ MPa} = 160 \times 10^6 \text{ Pa}$$

$$\sigma_{all} = \frac{M_x c}{I_x} \quad M_x = \frac{I_x \sigma_{all}}{c} = \frac{(105.271 \times 10^{-6})(160 \times 10^6)}{0.130} = 129.564 \times 10^3 \text{ N}\cdot\text{m}$$

$$M_x = 129.6 \text{ kN}\cdot\text{m}$$

4.6 Resolva o Problema 4.5 considerando que a viga seja flexionada em torno do eixo y por um momento fletor M_y .



For one flange, $I_f = \frac{1}{12} b_f h_f^3$
 $b_f = 16 \text{ mm}$ $h_f = 200 \text{ mm}$
 $I_f = \frac{1}{12} (16)(200)^3 = 10.6667 \times 10^6 \text{ mm}^4$

For the web $I_w = \frac{1}{12} b_w h_w^3$
 $b_w = 260 - (2)(16) = 228 \text{ mm}$ $h_w = 10 \text{ mm}$
 $I_w = \frac{1}{12} (228)(10)^3 = 19 \times 10^3 \text{ mm}^4$

Section: $I_y = 2I_f + I_w = 21.352 \times 10^6 \text{ mm}^4 = 21.352 \times 10^{-6} \text{ m}^4$

$c = \frac{260}{2} = 130 \text{ mm} = 0.130 \text{ m}$

$\sigma_{all} = \frac{\sigma_u}{F.S.} = \frac{400}{2.50} = 160 \text{ MPa} = 160 \times 10^6 \text{ Pa}$

$\sigma_{all} = \frac{M_y c}{I_y}$ $M_y = \frac{I_y c}{\sigma_{all}} = \frac{(21.352 \times 10^{-6})(160 \times 10^6)}{0.130}$
 $= 34.163 \times 10^3 \text{ N}\cdot\text{m}$

$M_y = 34.2 \text{ kN}\cdot\text{m}$ ◀

4.7 – 4.8 – 4.9

4.7 até 4.9 Duas forças verticais são aplicadas à viga com a seção transversal mostrada na figura. Determine as tensões de tração e de compressão máximas na parte BC da viga.

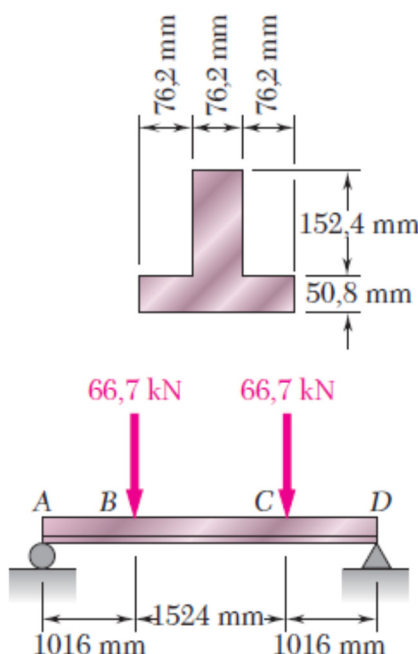


Fig. P4.7

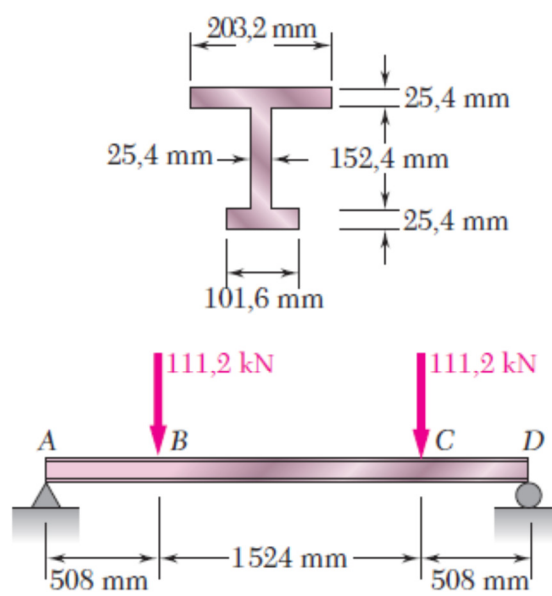


Fig. P4.8

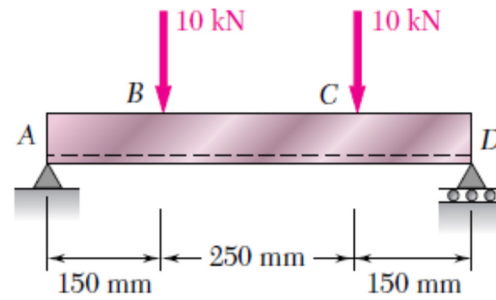
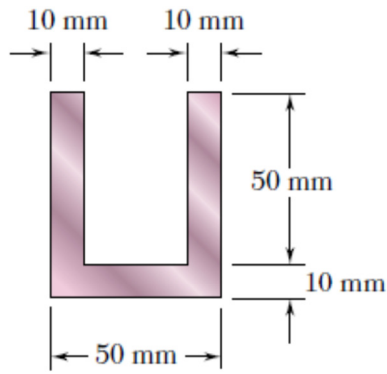
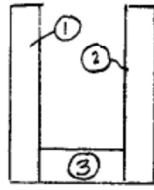
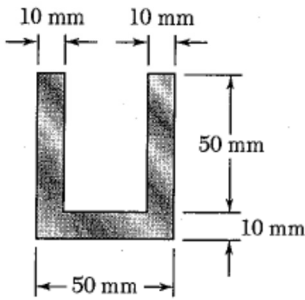
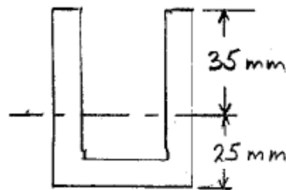
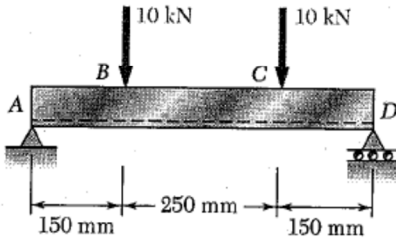


Fig. P4.9



	A, mm^2	\bar{y}_o, mm	$A\bar{y}_o, \text{mm}^3$
①	600	30	18×10^3
②	600	30	18×10^3
③	300	5	1.5×10^3
	1500		37.5×10^3

$$\bar{y}_o = \frac{37.5 \times 10^3}{1500} = 25 \text{ mm}$$



Neutral axis lies 25 mm above the base.

$$I_1 = \frac{1}{12}(10)(60)^3 + (600)(5)^2 = 195 \times 10^3 \text{ mm}^4$$

$$I_2 = I_1 = 195 \text{ mm}^4$$

$$I_3 = \frac{1}{12}(30)(10)^3 + (300)(20)^2 = 122.5 \times 10^3 \text{ mm}^4$$

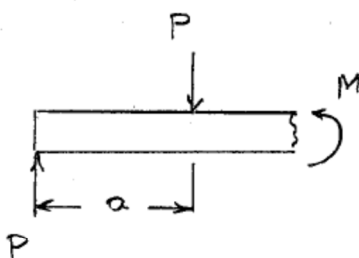
$$I = I_1 + I_2 + I_3 = 512.5 \times 10^3 \text{ mm}^4 = 512.5 \times 10^{-9} \text{ m}^4$$

$$y_{\text{top}} = 35 \text{ mm} = 0.035 \text{ m}$$

$$y_{\text{bot}} = -25 \text{ mm} = -0.025 \text{ m}$$

$$a = 150 \text{ mm} = 0.150 \text{ m} \quad P = 10 \times 10^3 \text{ N}$$

$$M = Pa = (10 \times 10^3)(0.150) = 1.5 \times 10^3 \text{ N}\cdot\text{m}$$



$$\sigma_{\text{top}} = -\frac{My_{\text{top}}}{I} = -\frac{(1.5 \times 10^3)(0.035)}{512.5 \times 10^{-9}} = -102.4 \times 10^6 \text{ Pa}$$

$$\sigma_{\text{top}} = -102.4 \text{ MPa}$$

(compression)

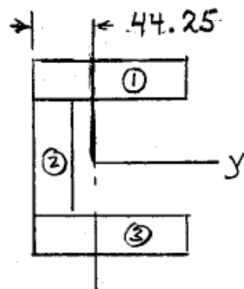
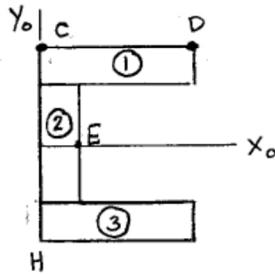
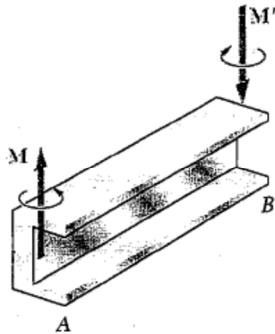
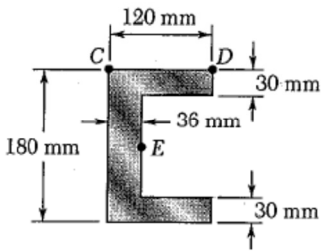
$$\sigma_{\text{bot}} = -\frac{My_{\text{bot}}}{I} = -\frac{(1.5 \times 10^3)(-0.025)}{512.5 \times 10^{-9}} = 73.2 \times 10^6 \text{ Pa}$$

$$\sigma_{\text{bot}} = 73.2 \text{ MPa}$$

(tension)

4.10

4.10 Dois momentos fletores iguais e opostos de intensidade $M = 25 \text{ kN} \cdot \text{m}$ são aplicados à viga AB constituída de um perfil U. Observando que fazem a viga flexionar em um plano horizontal, determine a tensão no (a) ponto C , (b) ponto D e (c) ponto E .



	A_i, mm^2	\bar{X}_0, mm	$A\bar{X}_0, \text{mm}^3$
①	3600	60	216×10^3
②	4320	18	77.76×10^3
③	3600	60	216×10^3
Σ	11520		509.76×10^3

$$\bar{X} = \frac{509.76 \times 10^3}{11520} = 44.25 \text{ mm}$$

$$y_c = -44.25 \text{ mm} = -0.04425 \text{ m}$$

$$y_D = 120 - 44.25 = 75.75 \text{ mm} = 0.07575 \text{ m}$$

$$y_E = 36 - 44.25 = -8.25 \text{ mm} = -0.00825 \text{ m}$$

$$d_1 = 60 - 44.25 = 15.75 \text{ mm}$$

$$d_2 = 44.25 - 18 = 26.25 \text{ mm}$$

$$d_3 = d_1$$

$$I_1 = I_3 = \frac{1}{12} b_1 h_1^3 + A_1 d_1^2 = \frac{1}{12} (30)(120)^3 + (3600)(15.75)^2 = 5.2130 \times 10^6 \text{ mm}^4$$

$$I_2 = \frac{1}{12} b_2 h_2^3 + A_2 d_2^2 = \frac{1}{12} (120)(36)^3 + (4320)(26.25)^2 = 3.4433 \times 10^6 \text{ mm}^4$$

$$I = I_1 + I_2 + I_3 = 6.5187 \times 10^6 \text{ mm}^4 = 13.8694 \times 10^{-6} \text{ m}^4$$

$$M = 15 \times 10^3 \text{ N}\cdot\text{m}$$

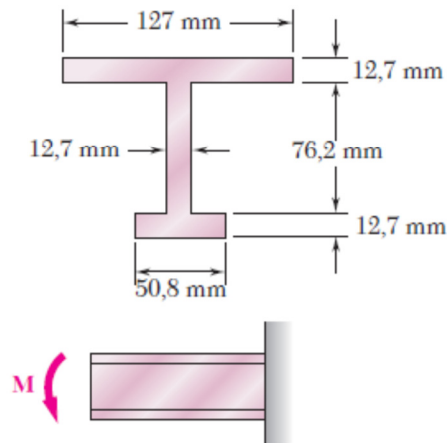
(a) Point C: $\sigma_c = -\frac{My_c}{I} = -\frac{(25 \times 10^3)(-0.04425)}{13.8694 \times 10^{-6}} = 79.8 \times 10^6 \text{ Pa}$
 $\sigma_c = 79.8 \text{ MPa}$ ▶

(b) Point D: $\sigma_D = -\frac{My_D}{I} = -\frac{(25 \times 10^3)(0.07575)}{13.8694 \times 10^{-6}} = -136.5 \times 10^6 \text{ Pa}$
 $\sigma_D = -136.5 \text{ MPa}$ ▶

(c) Point E: $\sigma_E = -\frac{My_E}{I} = -\frac{(25 \times 10^3)(-0.00825)}{13.8694 \times 10^{-6}} = 14.87 \times 10^6 \text{ Pa}$
 $\sigma_E = 14.87 \text{ MPa}$ ▶

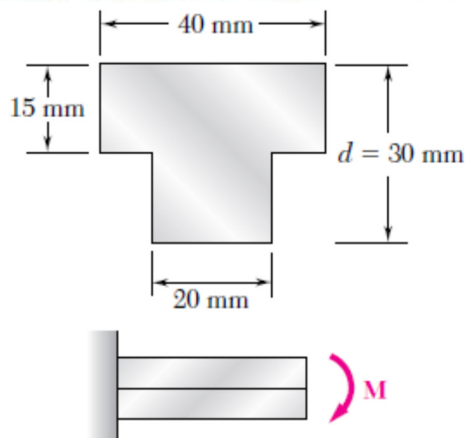
4.15

4.15 Sabendo que para a peça fundida mostrada na figura as tensões admissíveis na tração e na compressão são, respectivamente, $\sigma_{adm} = +41,4 \text{ MPa}$ e $\sigma_{adm} = -103,4 \text{ MPa}$, determine o maior momento fletor M que poderá ser aplicado.



4.16

4.16 A viga mostrada na figura é feita de um tipo de náilon para o qual a tensão admissível é de 24 MPa em tração e de 30 MPa em compressão. Determine o maior momento fletor M que pode ser aplicado à viga.



The neutral axis lies 17,5 mm above the bottom.

$$y_{top} = 30 - 17,5 = 12,5 \text{ mm} = 0,0125 \text{ m}, \quad y_{bot} = -17,5 \text{ mm} = -0,0175 \text{ m}$$

$$I_1 = \frac{1}{12} b_1 h_1^3 + A_1 d_1^2 = \frac{1}{12} (40)(15)^3 + (600)(5)^2 = 25,25 \times 10^3 \text{ mm}^4$$

$$I_2 = \frac{1}{12} b_2 h_2^3 + A_2 d_2^2 = \frac{1}{12} (20)(15)^3 + (300)(10)^2 = 35,625 \times 10^3 \text{ mm}^4$$

$$I = I_1 + I_2 = 61,875 \times 10^3 \text{ mm}^4 = 61,875 \times 10^{-9} \text{ m}^4$$

$$|\sigma| = \left| \frac{My}{I} \right| \quad M = \left| \frac{\sigma I}{y} \right|$$

$$\text{Top: tension side. } M = \frac{(24 \times 10^6)(61,875 \times 10^{-9})}{0,0125} = 118,8 \text{ N}\cdot\text{m}$$

$$\text{Bottom: compression. } M = \frac{(30 \times 10^6)(61,875 \times 10^{-9})}{0,0175} = 106,1 \text{ N}\cdot\text{m}$$

Choose smaller value.

$$M = 106,1 \text{ N}\cdot\text{m}$$

4.17

4.17 Resolva o Problema 4.16 considerando que $d = 40$ mm.

The neutral axis lies 23.41 mm above the bottom.

$$y_{top} = 40 - 23.41 = 16.59 \text{ mm} = 0.01659 \text{ m}$$

$$y_{bot} = -23.41 \text{ mm} = -0.02341 \text{ m}$$

$$I_1 = \frac{1}{12} b_1 h_1^3 + A_1 d_1^2 = \frac{1}{12} (40)(15)^3 + (600)(9.09)^2 = 60.827 \times 10^3 \text{ mm}^4$$

$$I_2 = \frac{1}{12} b_2 h_2^3 + A_2 d_2^2 = \frac{1}{12} (20)(25)^3 + (500)(10.91)^2 = 85.556 \times 10^3 \text{ mm}^4$$

$$I = I_1 + I_2 = 146.383 \times 10^3 \text{ mm}^4 = 146.383 \times 10^{-9} \text{ m}^4$$

$$|\sigma| = \left| \frac{M y}{I} \right| \quad M = \left| \frac{\sigma I}{y} \right|$$

Top: tension side. $M = \frac{(24 \times 10^6)(146.383 \times 10^{-9})}{0.01659} = 212 \text{ N}\cdot\text{m}$

Bottom: compression. $M = \frac{(30 \times 10^6)(146.383 \times 10^{-9})}{0.02341} = 187.6 \text{ N}\cdot\text{m}$

Choose smaller value.

$$M = 187.6 \text{ N}\cdot\text{m} \leftarrow$$

4.37

4.37 Três vigas de madeira e duas chapas de aço são aparafusadas firmemente para formar o elemento composto mostrado na figura. Usando os dados da tabela abaixo, determine o maior momento fletor admissível quando a viga é flexionada em torno do eixo horizontal.

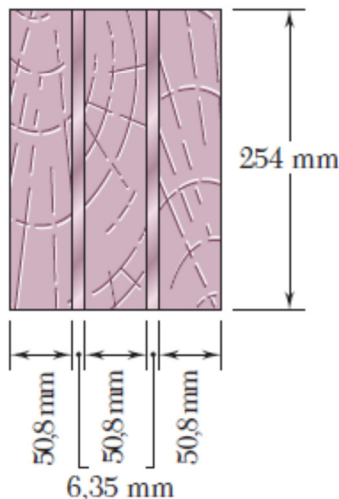


Fig. P4.37

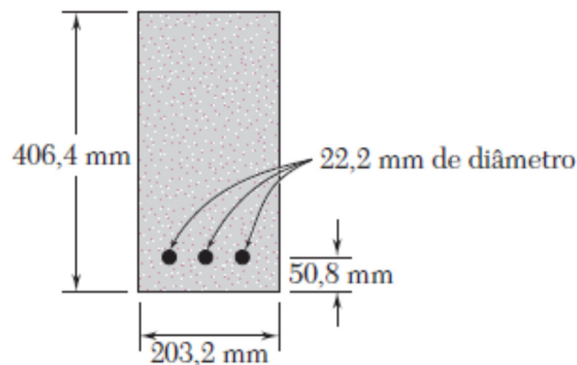
	Madeira	Aço
Módulo de elasticidade	13,8 GPa	207 GPa
Tensão admissível	13,8 MPa	152 MPa

4.38

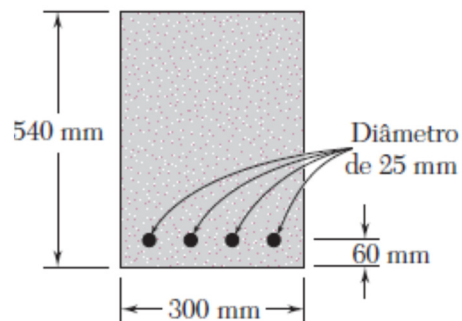
4.38 Para a viga composta da Fig. 4.37, determine o maior momento fletor admissível quando a viga é flexionada em torno do eixo vertical.

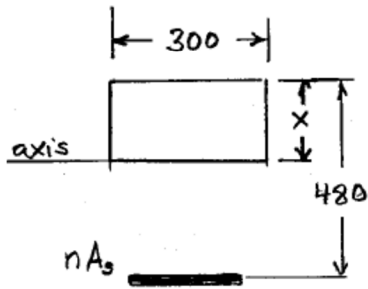
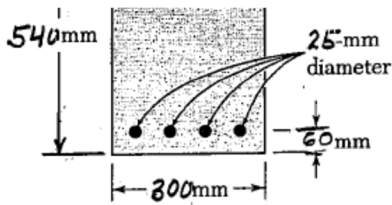
4.47

4.47 Uma viga de concreto é reforçada por três barras de aço colocadas conforme mostra figura. O módulo de elasticidade é de 20,7 GPa para o concreto e de 207 GPa para o aço. Usando uma tensão admissível de 9,31 MPa para o concreto e de 138 MPa para o aço, determine o maior momento fletor admissível positivo que pode ser aplicado à viga.

**4.48**

4.48 A viga de concreto reforçada mostrada na figura está submetida a um momento fletor positivo de $175 \text{ kN} \cdot \text{m}$. Sabendo que o módulo de elasticidade é de 25 GPa para o concreto e de 200 GPa para o aço, determine (a) a tensão no aço e (b) a tensão máxima no concreto.





$$n = \frac{E_s}{E_c} = \frac{200 \text{ GPa}}{25 \text{ GPa}} = 8.0$$

$$A_s = 4 \cdot \frac{\pi}{4} d^2 = (4) \left(\frac{\pi}{4} \right) (25)^2 = 1.9635 \times 10^3 \text{ mm}^2$$

$$nA_s = 15.708 \times 10^3 \text{ mm}^2$$

Locate the neutral axis.

$$300 \times \frac{x}{2} - (15.708 \times 10^3)(480 - x) = 0$$

$$150x^2 + 15.708 \times 10^3 x - 7.5398 \times 10^6 = 0$$

Solve for x.
$$x = \frac{-15.708 \times 10^3 + \sqrt{(15.708 \times 10^3)^2 + (4)(150)(7.5398 \times 10^6)}}{(2)(150)}$$

$$x = 177.87 \text{ mm}, \quad 480 - x = 302.13 \text{ mm}$$

$$\begin{aligned} I &= \frac{1}{3} 300 x^3 + (15.708 \times 10^3)(480 - x)^2 \\ &= \frac{1}{3} (300)(177.87)^3 + (15.708 \times 10^3)(302.13)^2 \\ &= 1.9966 \times 10^9 \text{ mm}^4 = 1.9966 \times 10^{-3} \text{ m}^4 \end{aligned}$$

$$\sigma = -\frac{nMy}{I}$$

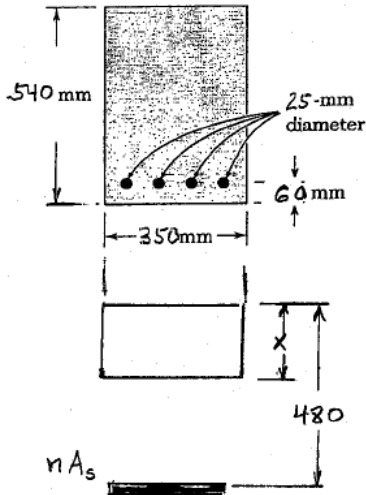
(a) Steel: $y = -302.45 \text{ mm} = -0.30245 \text{ m}$

$$\sigma = -\frac{(8.0)(175 \times 10^3)(-0.30245)}{1.9966 \times 10^{-3}} = 212 \times 10^6 \text{ Pa} = 212 \text{ MPa} \leftarrow$$

(b) Concrete: $y = 177.87 \text{ mm} = 0.17787 \text{ m}$

$$\sigma = -\frac{(1.0)(175 \times 10^3)(0.17787)}{1.9966 \times 10^{-3}} = -15.59 \times 10^6 \text{ Pa} = -15.59 \text{ MPa} \leftarrow$$

4.49 Resolva o Problema 4.48 considerando que a largura de 300 mm seja aumentada para 350 mm.



$$n = \frac{E_s}{E_c} = \frac{200 \text{ GPa}}{25 \text{ GPa}} = 8.0$$

$$A_s = 4 \frac{\pi}{4} d^2 = (4) \left(\frac{\pi}{4} \right) (25)^2 = 1.9635 \times 10^3 \text{ mm}^2$$

$$nA_s = 15.708 \times 10^3 \text{ mm}^2$$

locate the neutral axis

$$350 \times \frac{x}{2} - (15.708 \times 10^3)(480 - x) = 0$$

$$175x^2 + 15.708 \times 10^3 x - 7.5398 \times 10^6 = 0$$

Solve for x .

$$x = \frac{-15.708 \times 10^3 + \sqrt{(15.708 \times 10^3)^2 + (4)(175)(7.5398 \times 10^6)}}{(2)(175)}$$

$$x = 167.48 \text{ mm}, \quad 480 - x = 312.52 \text{ mm}$$

$$\begin{aligned} I &= \frac{1}{3}(350)x^3 + (15.708 \times 10^3)(480 - x)^2 \\ &= \frac{1}{3}(350)(167.48)^3 + (15.708 \times 10^3)(312.52)^2 \\ &= 2.0823 \times 10^9 \text{ mm}^4 = 2.0823 \times 10^{-3} \text{ m}^4 \end{aligned}$$

$$\sigma = -\frac{nMy}{I}$$

(a) Steel: $y = -312.52 \text{ mm} = -0.31252 \text{ m}$

$$\sigma = -\frac{(8.0)(175 \times 10^3)(-0.31252)}{2.0823 \times 10^{-3}} = 210 \times 10^6 \text{ Pa} = 210 \text{ MPa} \leftarrow$$

(b) Concrete: $y = 167.48 \text{ mm} = 0.16748 \text{ m}$

$$\sigma = -\frac{(1.0)(175 \times 10^3)(0.16748)}{2.0823 \times 10^{-3}} = -14.08 \times 10^6 \text{ Pa} = -14.08 \text{ MPa} \leftarrow$$

4.50

4.50 Uma laje de concreto é reforçada por barras de 15,88 mm de diâmetro colocadas com distanciamento de 139,7 mm entre os centros, conforme mostra a figura. O módulo de elasticidade é de 20,7 GPa para o concreto e de 200 GPa para o aço. Usando uma tensão admissível de 9,65 MPa para o concreto e de 137,9 MPa para o aço, determine o maior momento fletor por metro de largura que pode ser aplicado com segurança à laje.

